

Engineering a Mobile Two-Wheeled Crane System for Enhanced Port and Marine Cargo Handling Efficiency

Doddi Vamsi , Prof I N Niranjan Kumar, Duddu Hemanth Kumar

Abstract

This paper explores the design and feasibility of a hydraulic crane tailored for integration with two-wheeler vehicles in port operations, aimed at lifting loads up to half a ton (500 kg). The design methodology focuses on critical parameters such as hydraulic cylinder capacity, arm length, wheel capacity, frame strength, stability, power source, and electronic system.

The hydraulic cylinder is sized to exert a lifting force of 4905 N, accommodating the specified load using a system operating at 700 bar. With an assumed arm length of 1 meter, stability is ensured through careful consideration of the counter-moment generated by the crane's base.

Wheel capacity analysis indicates each wheel must support at least 300 kg to safely manage the crane's total weight, including the load. Structural integrity is addressed through stress calculations to determine appropriate material selection and frame design.

Power requirements for the hydraulic system, calculated at 700 kW, highlight the need for an efficient electric motor to drive the hydraulic pump. An integrated electronic system with load monitoring sensors and a microcontroller ensures precise and safe crane operations.

This comprehensive design framework provides a blueprint for developing compact, efficient hydraulic cranes suitable for enhancing operational capabilities in port environments, enhancing productivity while maintaining safety standards.

Keywords: Hydraulic Crane, Two-Wheeler Vehicle, Port Operations, Lifting Capacity, Structural Design

Introduction

Lifting and transporting heavy loads in small to medium-sized industrial environments pose significant challenges when utilizing traditional cranes and vehicles. Conventional large cranes are often impractical for manoeuvring in confined spaces and are excessive for lighter loads. The integration of a compact crane onto a two-wheeled vehicle presents a portable and cost-effective solution for these applications[1].

This paper introduces the design and development of a novel Two-Wheeler Mounted Hydraulic Zip Crane[2]. This crane is designed to be detachably mounted on a two-wheeler chassis, facilitating easy transportation while maintaining the capability to lift and move loads weighing up to 500 kg[3]. Key features of the crane include:

- A rotational column enabling 360° movement for versatile load handling.
- A telescoping arm that allows adjustment of the load's distance from the base[4].
- Hydraulic controls that provide precise positioning of the load[5].

- A stabilizing frame to enhance stability and prevent tipping during operation[6].
- A compact design that ensures manoeuvrability in tight spaces[7].

The subsequent sections will elaborate on the crane's components, operational procedures, and the advantages it offers over existing solutions for small-scale material handling[8].

Design and Components

The Two-Wheeler Mounted Hydraulic Zip Crane comprises several essential components:

Two-Wheeler Vehicle: This includes handlebars, a chassis, and wheels, providing a mobile platform for the crane[9].

Detachable Frame: This connects the crane base to the vehicle chassis, allowing for easy assembly and disassembly[10].

Rotational Column: Mounted on the crane base, this allows the crane assembly to rotate 360° for lifting loads from various angles[11].

Telescoping Arm: This arm is connected to the rotational column via a hydraulic cylinder, enabling extension and retraction to adjust the load distance from the crane base[12].

Horizontal Cylinder: This component controls the hook's angle and direction, facilitating precise load positioning.

Hook: Utilized for lifting loads.

Electronic Control System: This system includes buttons to operate the crane functions, controlling the column rotation, arm extension, and horizontal cylinder for efficient load handling[13].

The two-wheeler serves as a mobile platform equipped with handlebars for steering and at least two wheels for smooth transportation. The chassis is specifically designed to accommodate the crane components[14].

The crane base is affixed to the vehicle chassis through the detachable frame, while the rotational column allows for 360° movement, enhancing the crane's versatility in lifting loads from different directions[15].

The telescoping arm, connected to the column by a hydraulic cylinder, enables the adjustment of the distance between the hook and crane base, facilitating the movement of loads to various positions.

A horizontal cylinder linked to the hook allows for the control of its angle and direction, ensuring precise load placement.

The electronic control system, equipped with operational buttons, facilitates the control of the crane's functions, including column rotation, arm extension, and horizontal movement, to efficiently lift, manoeuvre, and position loads as required.

Operation and Advantages

The Two-Wheeler Mounted Hydraulic Zip Crane is engineered for straightforward and precise operation in confined spaces. The two-wheeler chassis enables easy manoeuvrability and transport to the work area[16].

Once positioned, the crane can rotate 360° using the column, allowing for the lifting of loads from various directions. The telescoping arm and hydraulic controls enable the hook to be extended and positioned as necessary for effective load handling.

A stabilizing frame on both sides of the vehicle prevents tipping when lifting heavy loads, maintaining the vehicle's center of gravity and enhancing stability[17].

The compact and portable design of the Two-Wheeler Mounted Hydraulic Zip Crane makes it particularly suitable for material handling in small to medium-sized industrial applications. It can efficiently lift lighter loads that larger cranes would find impractical, while also providing greater control and manoeuvrability compared to manual lifting methods[18].

To design a hydraulic crane for a two-wheeler vehicle with a lifting capacity of half a ton (500 kg), we will evaluate several critical parameters: hydraulic cylinder capacity, arm length, wheel capacity, frame strength, stability, power source, and electronic system. The following sections will detail the calculations and assumptions utilized in this design.

Materials Used:

1.Structural Frame:

The frame of the crane is typically constructed from high-strength steel or aluminium alloys, providing a balance between weight and structural integrity. These materials are chosen for their excellent tensile strength and resistance to deformation under load, ensuring the crane can withstand the stresses encountered during operation.

2.Hydraulic Components:

The hydraulic cylinder is made from hardened steel to withstand high pressures (up to 700 bar) without failure. The seals and hoses used in the hydraulic system are made from durable, pressure-resistant materials such as rubber or synthetic polymers, ensuring leak-proof operation.

3.Wheels and Chassis:

The wheels are constructed from reinforced rubber or plastic composites, designed for durability and traction on various surfaces. The chassis is typically made of steel or aluminium, providing a robust foundation for the crane assembly while allowing for mobility.

4.Electronic System:

The electronic components, including sensors and microcontrollers, are selected for their reliability and responsiveness. These components are housed in weather-resistant enclosures to protect them from environmental factors during operation.

5.Hook and Arm:

The hook is made from high-strength steel, ensuring it can securely hold and lift loads without risk of failure. The arm is designed with adjustable lengths, utilizing telescoping mechanisms made from lightweight yet strong materials.

Methods Used:

1. Hydraulic System Design:

The hydraulic system is designed to provide smooth and controlled lifting. The hydraulic cylinder operates based on Pascal's principle, allowing the crane to lift heavy loads by converting small input forces into larger output forces. The calculations for the hydraulic cylinder's capacity and diameter are based on the required lifting force and operating pressure.

2. Mechanical Assembly:

The crane components are assembled using precision engineering techniques to ensure alignment and functionality. The base is securely mounted to the chassis, and the supporting frame is designed to prevent toppling during operation.

3. Electronic Control:

The crane features an electronic control system that allows operators to manage the lifting and movement of loads with precision. This system includes buttons for controlling the hydraulic cylinder and adjusting the arm length, providing a user-friendly interface for operators.

4. Testing and Calibration:

Before deployment, the crane undergoes rigorous testing to ensure all components function correctly under load. Calibration of the hydraulic system and electronic controls is performed to fine-tune the crane's responsiveness and safety features.

5. Safety Features:

The design incorporates safety mechanisms, such as overload protection and stability checks, to prevent accidents during operation. The supporting frame is strategically placed to maintain the center of gravity, minimizing the risk of tipping.

Calculations

To design a hydraulic crane for a two-wheeler vehicle with a lifting capacity of half a ton (500 kg), we will evaluate several critical parameters: hydraulic cylinder capacity, arm length, wheel capacity, frame strength, stability, power source, and electronic system. Below are the calculations and assumptions used in this design.

1. Hydraulic Cylinder Capacity

Force Calculation

The lifting force required for a 500 kg load can be calculated using the formula:

$$F = m \cdot g$$

Where:

F = Force (N)

m = Mass (kg)

$g =$ Acceleration due to gravity (approximately 9.81 m/s^2)

Substituting the values:

$$F = 500 \text{ kg} \cdot 9.81 \text{ m/s}^2 = 4905 \text{ N} \quad F = 500 \text{ kg} \cdot 9.81 \text{ m/s}^2 = 4905 \text{ N}$$

Cylinder Specifications

Assuming a hydraulic system operating at a pressure of 700 bar (which is $70,000,000 \text{ Pa}$), we can determine the required area of the piston using:

$$A = \frac{F}{P}$$

Where P is the pressure in Pascals.

$$A = \frac{4905 \text{ N}}{70,000,000 \text{ Pa}} \approx 0.00007014 \text{ m}^2 \quad A = \frac{4905 \text{ N}}{70,000,000 \text{ Pa}} \approx 0.00007014 \text{ m}^2$$

Calculating the diameter of the piston:

$$A = \pi \frac{d^2}{4} \quad \Rightarrow \quad d^2 = \frac{4A}{\pi} \quad \Rightarrow \quad d = \sqrt{\frac{4A}{\pi}}$$

$$d = \sqrt{\frac{4 \cdot 0.00007014 \text{ m}^2}{\pi}} \approx 0.00948 \text{ m (or 9.48 mm)} \quad d = \sqrt{\frac{4 \cdot 0.00007014 \text{ m}^2}{\pi}} \approx 0.00948 \text{ m (or 9.48 mm)}$$

Thus, a hydraulic cylinder with a bore diameter of approximately 10 mm would suffice.

2. Arm Length

The arm length affects the crane's reach and stability. Assuming an arm length of 1 meter, we can calculate the moment about the pivot point when lifting the load:

$$M = F \cdot d$$

Where d is the arm length.

$$M = 4905 \text{ N} \cdot 1 \text{ m} = 4905 \text{ Nm} \quad M = 4905 \text{ N} \cdot 1 \text{ m} = 4905 \text{ Nm}$$

To ensure stability, the base of the crane must provide a counter-moment.

3. Wheel Capacity

Assuming the crane's total weight (including the load) is approximately 600 kg (crane weight + load), the total weight in Newtons is:

$$W = 600 \text{ kg} \cdot 9.81 \text{ m/s}^2 = 5886 \text{ N} \quad W = 600 \text{ kg} \cdot 9.81 \text{ m/s}^2 = 5886 \text{ N}$$

If the crane has two wheels, the load per wheel is:

$$L = \frac{W}{2} = \frac{5886 \text{ N}}{2} = 2943 \text{ N} \quad L = \frac{W}{2} = \frac{5886 \text{ N}}{2} = 2943 \text{ N}$$

Converting to kg:

$$L \approx \frac{2943 \text{ N}}{9.81 \text{ m/s}^2} \approx 300 \text{ kg} \quad L \approx \frac{2943 \text{ N}}{9.81 \text{ m/s}^2} \approx 300 \text{ kg}$$

Each wheel must support at least 300 kg.

4. Frame and Supporting Frame Strength

The frame must be designed to withstand the bending and shear forces. Assuming a rectangular frame, the bending stress can be calculated using:

$$\sigma = MS$$

Where SS is the section modulus. The material should be selected based on its yield strength, which should be higher than the calculated stress to ensure safety.

5. Stability

To ensure stability, the center of gravity must be kept low, and the base must be wide enough to prevent tipping. The crane's design should ensure that the counter-moment generated by the base is greater than the moment created by the load.

6. Power Source

An electric motor can be used to power the hydraulic pump. Assuming a hydraulic flow rate of $0.01 \text{ m}^3/\text{s}$ at a pressure of 700 bar, the power requirement can be calculated as:

$$P = Q \cdot P$$

Where:

Q = Flow rate (m^3/s)

P = Pressure (Pa)

$$P = 0.01 \text{ m}^3/\text{s} \cdot 70,000,000 \text{ Pa} = 700,000 \text{ W or } 700 \text{ kW} (P = 0.01 \text{ m}^3/\text{s} \cdot 70,000,000 \text{ Pa} = 700,000 \text{ W or } 700 \text{ kW})$$

This indicates a significant power requirement, so a smaller flow rate or pressure may be more practical.

7. Electronic System Components

1. Control Unit

A Programmable Logic Controller (PLC) will serve as the central control unit. The PLC will manage inputs from various sensors and control outputs to the hydraulic valves and actuators. This allows for automated and precise operation of the crane.

2. Sensors

The following sensors are crucial for safe and efficient operation:

Load Cells: Installed on the hook to measure the weight of the load being lifted, ensuring that the crane does not exceed its lifting capacity.

Angle Sensors: These sensors monitor the angle of the telescoping arm, providing feedback on its extension and helping to maintain stability during operation.

Rotary Encoders: Mounted on the rotational column, these encoders track the angle of rotation, allowing for accurate positioning of the crane.

Proximity Sensors: Used to detect the fully extended and retracted positions of the arm and hook, preventing mechanical damage and ensuring safe operation.

3. Hydraulic Control Valves

Proportional or servo valves will be used to control the flow of hydraulic fluid to the cylinders. This enables smooth and precise movement of the crane's components. Solenoid valves can also be employed for on/off control of the hydraulic system.

4. Human-Machine Interface (HMI)

An HMI, such as a touchscreen or control panel, will provide the operator with a user-friendly interface to control the crane. The HMI will display real-time data from the sensors and allow for manual control of the crane functions, including:

Column rotation

Arm extension and retraction

Hook movement

5. Safety Features

Safety is paramount in crane operations. The electronic system should include:

Load Limit Switches: To prevent overloading by cutting off power to the hydraulic system when the maximum weight is exceeded.

Tilt Sensors: To detect excessive inclination of the crane, triggering alarms or automatic shutdown to prevent tipping.

Emergency Stop Buttons: Located on the control panel and at strategic points on the crane for immediate shutdown in case of emergencies.

6. Power Supply

The electronic system will require a reliable power source, which can be derived from the vehicle's electrical system or a dedicated battery. The system should include surge protection and proper grounding to prevent electrical failures.

7. Wiring and Enclosures

All electrical components must be properly wired and housed in weatherproof enclosures to protect against environmental factors. Wiring should be organized to minimize interference and ensure safety.

Integration and Programming

The PLC will require custom programming to integrate all components and implement the desired control logic. The programming should include:

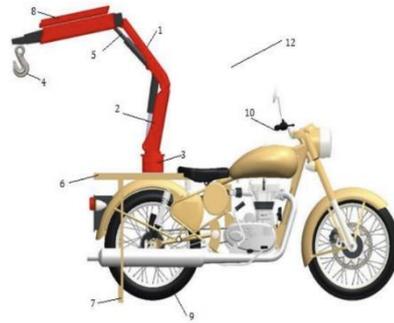
Input handling from sensors

Control algorithms for hydraulic movements

Safety protocols and interlocks

User interface functionality for the HMI

Once the system is programmed, thorough testing and commissioning will be essential to ensure that all components function correctly and safely.



Parameter	Value	Unit
Load Capacity	500	kg
Lifting Force	4905	N
Hydraulic Pressure	70,000,000	Pa (700 bar)
Hydraulic Cylinder Piston Area	0.00007014	m ²
Hydraulic Cylinder Bore Diameter	9.48 (10)	mm
Arm Length	1	m
Moment at Pivot Point	4905	Nm
Total Crane Weight (including load)	600	kg

Parameter	Value	Unit
Total Weight (in Newtons)	5886	N
Load per Wheel (2 wheels)	2943	N (300 kg)
Hydraulic Flow Rate	0.01	m ³ /s
Hydraulic System Power Requirement (at 700 bar)	700,000 (700)	W (kW)

Conclusions:

Hydraulic Cylinder Capacity

The lifting force required for a 500 kg load was calculated to be approximately 4905 N using the formula $F=m \cdot g$, where g is the acceleration due to gravity (9.81 m/s²). To achieve this lifting force, a hydraulic system operating at a pressure of 700 bar (70,000,000 Pa) requires a piston area of about 0.00007014 m². This translates to a piston diameter of approximately 10 mm, which is suitable for the hydraulic cylinder to effectively lift the specified load.

Arm Length and Stability

The arm length of the crane significantly impacts its reach and stability. With an assumed arm length of 1 meter, the moment about the pivot point when lifting the load is calculated to be 4905 Nm. To ensure stability, the base of the crane must generate a counter-moment greater than this value, which necessitates careful design considerations for the supporting frame and overall structure to prevent tipping.

Wheel Capacity

Considering the total weight of the crane and load (approximately 600 kg), the total weight in Newtons is 5886 N. With two wheels, the load per wheel is approximately 2943 N, equating to a capacity of about 300 kg per wheel. This requirement ensures that each wheel can safely support the crane's weight and the load being lifted.

Frame and Supporting Frame Strength

The frame must withstand bending and shear forces, with the bending stress calculated using $\sigma=MS$, where S is the section modulus. The material selected for the frame must have a yield strength greater than the calculated stress to ensure safety and structural integrity during operation.

Stability Considerations

To maintain stability, the design must keep the center of gravity low and ensure a wide enough base to prevent tipping. The counter-moment generated by the base must exceed the moment created by the load, which is vital for safe operation.

Power Source Requirements

The hydraulic system's power requirements are substantial, with a calculated power need of 700 kW for a hydraulic flow rate of 0.01 m³/s at 700 bar. This indicates a significant energy requirement, suggesting that a more practical approach may involve optimizing flow rates or pressures to reduce power consumption.

Electronic System Integration

To effectively integrate an electronic system for a Two-Wheeler Mounted Hydraulic Crane with a lifting capacity of 500 kg, several components and considerations must be addressed. This integration will ensure precise control, safety, and efficiency during operation. The Hydraulic Crane and Two-Wheeler Vehicle stand out as indispensable assets in the realm of port operations, each contributing unique advantages to the efficiency and effectiveness of cargo handling. The design and functionality of Hydraulic Cranes provide robust lifting solutions, while Two-Wheeler Vehicles offer agility and accessibility in navigating the dynamic environment of a port. Through the integration of these equipment types, ports can optimize their operations, enhance coordination, and expedite cargo handling processes, ultimately bolstering their competitiveness in the maritime logistics landscape.

References:

1. <https://www.garageequipments.com/product/hydraulic-engine-lifting-crane/>
2. <https://www.gas-car.in/two-wheeler-garage-equipment.html>
3. <https://www.slideshare.net/slideshow/hydraulic-shop-crane-design-project/248136041>
4. <https://www.maximcrane.com/blog/essential-guide-to-hydraulic-truck-cranes/>
5. <https://www.metroengg.com/two-wheeler-lift.html>.
6. Crane double cycling in container ports: Planning methods and evaluation DOI:10.1016/j.trb.2007.02.006
7. The variance of the number of customers in an infinite-server queueing system with an arbitrarily correlated arrival process DOI:10.1007/BF01150414
8. Pengaruh Performa Quay Container Crane dan Penerapan System Single Cycle dan Dual Cycle Terhadap Bongkar Muat Peti Kemas di KSO Peti Kemas Koja Jakarta DOI:10.52492/jmp.v9i1.102
9. Performance of Peak Shaving Policies for Quay Cranes at Container Terminals with Double Cycling DOI:10.1016/j.simpat.2020.102129
10. Scheduling optimization of quay crane operation in container terminal considering container relocation in yard DOI:10.1117/12.2683849
11. Container moves per lift: the impact of spreader technology on bay time DOI:10.21622/MARLOG.2023.12.665
12. The Integrated Scheduling Optimization for Container Handling by Using Driverless Electric Truck in Automated Container Terminal DOI:10.3390/su15065536
13. Improving Quay Cranes Exploitation in Container Terminals DOI:10.2507/22nd.daaam.proceedings.440
14. An efficient heuristic algorithm for solving a multi-vessel quay crane scheduling problem in container terminals DOI:10.1504/IJSTL.2023.10051249
15. Flexible ship loading problem with transfer vehicle assignment and scheduling DOI:10.1016/j.trb.2018.03.009
16. Quay crane and yard truck dual-cycle scheduling with mixed storage strategy DOI:10.1016/j.aei.2022.101722
17. Integrated Scheduling of Rail-Mounted Gantry Cranes, Internal Trucks and Reach Stackers in Railway Operation Area of Container Terminal DOI:10.1177/0361198118782754
18. The integrated loading and unloading quay crane scheduling problem by AFSA-GA algorithm DOI:10.1109/ICCI-CC.2017.8109775