

## **VARIATION OF TWO BODY ABRASIVE WEAR AND THREE BODY ABRASIVE WEAR IN ALUMINIUM ALLOY (6063)**

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### **Abstract**

This study aims to correlate the abrasive wear performance with mechanical properties, considering AA6063 Al-Mg-Si alloy as the model material. The selected alloy specimens are subjected to artificial ageing at 150 °C for an ageing duration ranging from 1 to 672 h, covering severely under-aged (SUA) to peak-aged (PA) to severely over-aged (SOA) states. Apart from the hardness and tensile properties, two-body abrasive wear properties are also evaluated for differently aged alloys in terms of wear rate, coefficient of friction, and roughness of the abraded surfaces. Furthermore, the generated wear debris, surface, and sub-surface of the abraded specimens are critically examined to reveal the micro-mechanisms of abrasion. The lowest amount of wear rate is observed for a PA alloy with maximum hardness, while the OA alloy exhibits a slightly lower wear rate than the UA alloy at a similar level of hardness. Statistical analyses of wear rate and various mechanical properties of all heat-treated alloys establish a strong negative linear correlation between the wear rate and hardness, yield strength, tensile strength, and strength coefficient; whereas, a positive linear correlation with the strain hardening exponent. Relationships between wear rate and different roughness parameters are also discussed. Under the investigated wear condition, the aged alloys endure significant plastic deformation; micro-plowing, micro-cutting, and delamination are found to be the predominant mechanisms during abrasion.

## 1. Introduction

The emission of greenhouse gases and global warming are the potential threats to the sustainable development of the human race. Hence, from the last few decades, continuous efforts are being made to cut down the fuel consumption and the emission rate of greenhouse gases in transportation sectors; this has led to the emergence of light alloys, *e.g.*, aluminium, magnesium, and titanium alloys, for structural and rotating components in automobile and aerospace industries [1–4]. Age-hardenable aluminium alloys are the most preferred ones in this field due to their excellent ageing response, high specific strength, and good corrosion resistance [5–8]. Al-Mg-Si alloys, especially the low Cu content members, are currently the economical ones employed for these applications [9]. The progress of age-hardening depends on the amount of added trace elements which can precipitate as per the kinetics of the alloy [10,11]. Additionally, Al-Mg-Si alloys undergo several microstructural alterations during the process of ageing; such microstructural changes have profound effects on the mechanical properties [12].

One of the significant drawbacks of Al alloys in the transportation sectors is their poor wear performance [13,14]. Several investigators [15–20] have attempted to understand the wear behavior of age-hardenable Al-alloys. Reports related to the wear of Al-alloys are primarily directed at understanding the adhesive wear behavior [15,16], although the frequently encountered mode of wear during industrial applications is abrasion. Wear causes damage to the material surface when it comes in contact with other materials like solid, liquid, and gas [21]. Abrasive wear removes the material on the surface in the form of debris when a hard surface slides against a softer surface under normal loading conditions [22]. Earlier investigations related to the abrasive wear behavior of Al-Mg-Si are primarily focused on the effect of Si content [17] and the temperature of ageing [16]. Kaushik et al. [23,24] have investigated the influence of applied pressure and the abrasive grit size on the high-stress abrasive wear resistance of hybrid 6082 Al-Mg-Si composites and compared them with the unreinforced alloy in both as-cast as well as in T6 conditions. The authors have observed that the abrasive wear resistance is the highest under lower applied pressures and grit size and gradually decreases with increasing the same for both as-cast and the aged conditions. At the same time, comparatively, the aged specimen has shown higher abrasion resistance at any particular applied pressure and grit sizes.

However, from the tribological perspective, besides understanding the type of wear encountered, wear behavior, and their mechanisms, studying the correlation between the wear and mechanical properties are also equally important in enhancing the entire life cycle of mechanical components by defining the abrasive wear losses [5]. Few earlier researchers have attempted to correlate the abrasive wear performance with mechanical properties of materials. Baldoni et al. [25] have examined the abrasive wear resistance of three different classes of cutting tool materials (brittle, brittle-ductile, and ductile) in correlation with the hardness and fracture toughness. Wear by fracture is dominant for the brittle materials, which indicates the abrasive wear resistance depends on the fracture toughness of the material. For brittle-ductile material like cemented carbide, wear can cause a

large amount of plastic deformation, which is less prone to fracture toughness; whereas, for ductile metals, low hardness and fracture toughness results in less abrasive wear resistance. Jha et al. [26] have studied the correlation between the hardness, microstructure, and tensile properties of high-strength low alloy steel. These researchers have optimized the microstructural features and other mechanical properties to achieve superior wear characteristics and suggested that apart from hardness, ductility is also equally important in defining the abrasive wear performance of the steel. In a different study, Das et al. [27] have investigated the wear resistance of the aluminium alloys and hard particle composites. This research group has established an equation to define the wear rate as a function of hardness, strength, and young's modulus and has suggested that the wear resistance is being entirely dependent on hardness, even though other mechanical properties like strength and elongation have a minor influence on it.

One can find a gap in the existing literature regarding the correlation between the abrasive wear characteristics and the mechanical properties of materials in contact. This study focuses on understanding two-body abrasive wear behavior and its correlation with the mechanical properties of artificially aged AA6063 alloy. Apart from establishing the hardness and tensile properties, the two-body abrasive wear behavior is examined in terms of wear rate, coefficient of friction, and roughness of the abraded surfaces. The operative wear mechanisms are identified by characterizing the abrasive paper, generated debris, surfaces, and sub-surfaces of abraded specimens using SEM and EDS apart from surface topography using a 3D optical profilometer. Finally, an attempt to correlate the wear rate with mechanical properties and some roughness parameters is made for Al-Mg-Si alloy specimens subjected to different ageing conditions.

## REVIEW OF LITERATURE

- **Saurabh Kumar [1]** published three body abrasive wear behaviour on six aluminium alloys (AA1050, AA2014-T6, AA3003, AA5052, AA6061-T6 and AA6351-T6). Research shows abrasive wear increases with increase in load. Interaction of abrasive wear rate with hardness, ultimate strength and percentage elongation at break are reported. Wear mechanisms involved in material removal process were studied with the aid of optical microscope and scanning electron microscope (SEM).
- **Monil Salot [2]** This study focuses on the evaluation of abrasive wear resistance of Al-Si alloys with and without chromium oxide coating. It states that the wear resistance of Al-Si alloys increases with the increase in the amount of Si in the alloy.
- **Tadashi Ohtani [3]** Two-body wear and three-body abrasive wear test of katsura wood were carried out using abrasive paper and moving abrasive grains. It is reported that two-body abrasive wear is more affected by yield stress and surface microstructure, and three-body abrasive wear is more affected by the cutting action of moving abrasive grains.
- **A.P. Harsha [4]** Investigates the three-body abrasive wear behavior of polyaryletherketone composites. It includes the study the effect of reinforcement fibres, solid lubricants, mass of abrasives and load in three-body abrasive wear situations on various polyaryletherketone (PAEK) matrix. He inspected that fibre reinforcement is detrimental to the abrasive wear resistance of neat PAEK matrix. Scanning electron microscopy was used to detect worn surfaces and to know the functioning involved in it.
- **Mohd Shadab Khan [5]** conducted a research on Abrasive wear behavior of Silicon Polyethylene vinyl Acetate copolymer using a pin-on-disc arrangement. Load and angular speed are the two

parameters which are taken into consideration .Results showed that the wear increases as the load and angular speed(rpm).

- **Mohd shadab Khan** [6] inspected the Abrasive wear behavior of Organopolysiloxane and concluded that the wear varies with applied load and and mechanical parameters .The wear increases with increasing load and angular speed.
- **N. Mohan** [7] investigates the Two-Body Abrasive Wear Behavior of Silicon Carbide Filled Glass Fabric-Epoxy Composites. The effect of silicon carbide (SiC) fillers inclusion on two-body abrasive wear behaviour of Glass Fabric - Epoxy (GE) composites was observed and results are examined. He states that the appreciably decrement in wear loss and specific wear rates were observed after introducing of SiC filler into GE composites.
- **Gaoqi Wang** [8] examined the Two-Body and Three-Body Wear Behavior of a Dental Fluorapatite Glass-Ceramic. Results obtained that good mechanical properties of fluorapatite glass-ceramic can be achieved by the sintering process. In both two-body and three-body modes, the fluorapatite glass-ceramic had a smaller friction coefficient and wear rate and caused less damage on antagonistic teeth than the feldspathic glass-ceramic.
- **Vishwas Mahesh** [9] performed research on two-body and three-body abrasive wear behavior of jute-natural rubber flexible green composite. The wear trend of the composites follows a similar pattern in the case of two- and three-body wear, the mechanisms governing the wear are found to be different. The morphology of the worn surface is studied with the aid of a scanning electron microscope.

## EXPERIMENTAL PROCEDURES

### Material

### Experimentation

In this chapter, details of material used in the present investigation and its preparation has been described and the details of the experimentation on wear studies in the material of present investigation have been given.

In order to carry out the experimental work, the procedure is as follows.

- (i) Fabrication of Pin on Disc
- (ii) Specimen's Materials
- (iii) Wear characterization

### Fabrication of Pin on Disc

Numerous researches have been done in the field of abrasive wear. This work is also an experimental design in the field of abrasive wear Pin on Disc via a newly designed wear Pin on Disc. In view of the objective a set-up was needed to be designed which can calculate wear rate at different speed (rpm) of work piece with respect to the main frame (horizontal position).

The wear machine used for Pin on Disc wear properties was designed by **Prof. (Dr.) Zahir Hasan** and fabricated by **Dr. Mohd Shadab Khan**. A pin on disc wear test technique was adopted to test the wear behavior of specimens.

Wear rate and wear mass were Pin on Disc at different orientation of the specimen. The tests were conducted for seven different orientations namely **100 rpm, 150 rpm, 200 rpm**. The wear mass of above said specimen Pin on Disc test at a constant time of **2min (120 sec)**.

The set-up has following different parts:-

(1)Controller (2)D.C Motor (3)Flange Coupling (4)Bearing (5)Main Frame (6)Frame(Angular) (7)AcrylicSheet (8) Grinding Wheel (9) Specimen (10) Screw Jack (11) Load Cell (12) Angular Lever.

The designed setup is shown in the fig. 4.1 and 4.2

### Experimental Setup of Wear Pin on Disc

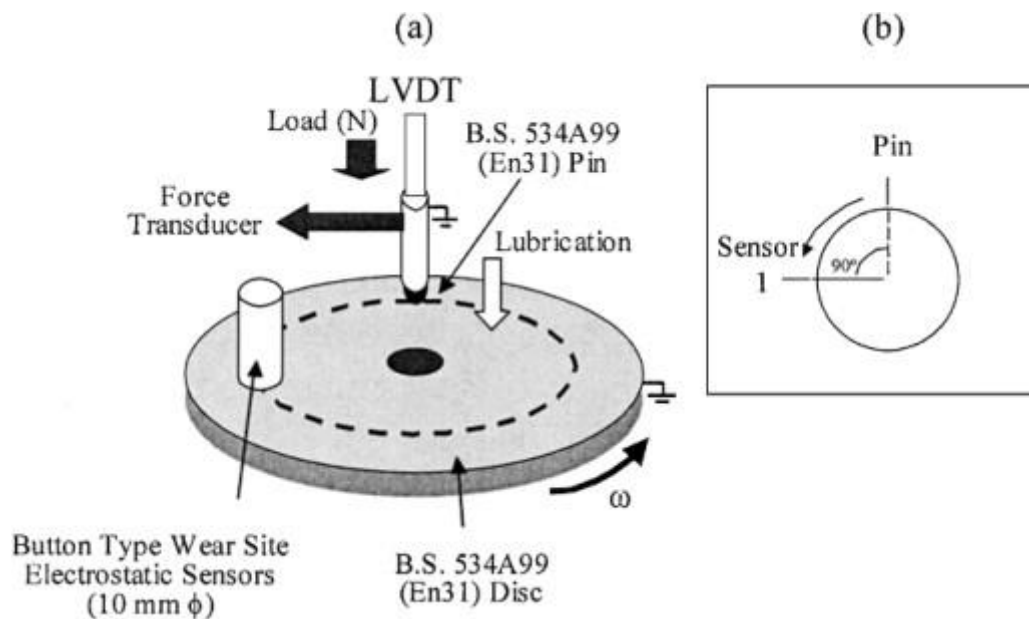


Fig- 4.1 Experimental Setup (Front View)

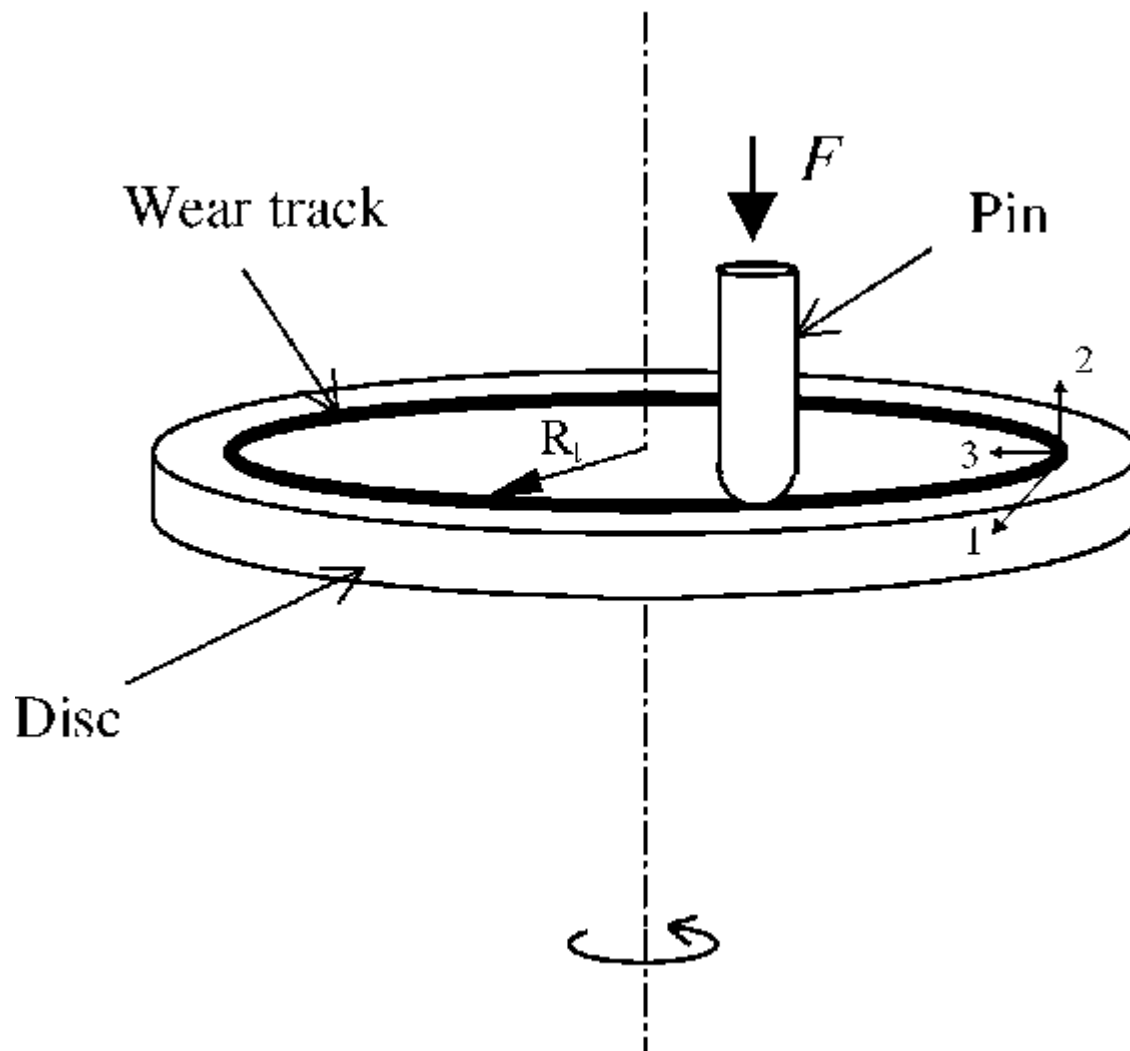


Fig- 4.1 Experimental Setup (Top View)



## **Working of Set-Up**

The pin-on-disc sliding tests were conducted in the following manner. The pins, measuring 5 mm in diameter and 5 mm in length, were made of hardened bearing steel (AISI 52100) and polished to a surface roughness of Ra 0.05 micron. The disc measured 35 mm in diameter and 2.5 mm in thickness and was made of carburized steel. The three pins were secured to prevent them from rotating and were pressed against the toric sliding surface of the rotating disc at a position that was 20 mm in diameter from the center of the disc. Contact at the sliding interfaces was in the shape of lines under high Hertzian pressure of 700 MPa due to a normal force of 500 N, Lubrication was provided by an oil bath heated to 353 K. The sliding speed was varied in a range of 0 to 1 ms<sup>-1</sup> for the tribological experiments and the sliding time was 60 minutes..

## **Description of the Parts of the Wear Pin on Disc**

### **DC Motor**

The D.C. Motor having following specifications:

Power – 1 H.P, Rotation – 1 rpm to 3000 rpm

Regulator of a direct current motor is used to regulate and control the speed of motor. It has ammeter to measure current and voltmeter to measure volt attached to it. The characteristic features of regulator are:

Regulated Voltage – 0-260 V , Least Count – 2V The technical parameters of Ammeter are:Current

Range: 0 – 10 ampere

Least Count: 0.4 ampere

The technical parameters of Voltmeter are: Voltage Range : 0 – 300V , Least Count: 20 V

## Frame

The main frame is just like chassis to the engine ,it hold all the parts such as motor , shaft , coupling , screw jack and all its related attachment. The dimensions of the main frame are as follows:

Length – 105 cm , Width – 21 cm Dimensions of angular frame are as follows :Length – 115 cm Width – 35 cm

## Acrylic Disc

The acrylic disc is used as a fixture of specimen holder. The disc is drilled with multiple holes at different radius. This is done so that every specimen gets fresh abrasive surface. This makes synchronization in the calculation of wear rate of the entire specimen. The dimensions of the acrylic disc are as follows:

Diameter – 26 cm

Radius of the first hole ( $r_1$ ) = 8 cm Radius of the second hole ( $r_2$ ) = 16 cm Radius of the third hole ( $r_3$ ) = 24 cm

## Grinding Wheel

A grinding wheel used in the design as an abrasive media to produce abrasive wear on the specimen selected. The dimensions of the grinding disc are as follows:

Diameter – 20 cm

## Speed Of Grinding Wheel

Generally all the abrasive processes are performed with the wheel speed in between the range of 300 to 2000 rpm with the maximum work speed from 0 to 60 m/min.

## Shaft

Two shafts were used, the first shaft connects motor to the abrasive disc and second shaft connects acrylic disc to screw jack. Load is applied with help of second shaft, it pushes the specimen against the rotating abrasive disc.

The dimensions of the shaft are as follows :

Diameter – 25mm Length (First) – 20 cm Length (Second) – 30 cm

## Screw Jack

The screw jack is used to apply the load gradually turn wise. The screw jack is connected to the shaft, which is further connected to the acrylic disc and specimen fixture. As the screw jack unfolds, it pushes the shaft and acrylic sheet which holds the specimen against the abrasive disc.

## Weighing Machine

The weighing machine used in the design to calculate mass loss (wear) of the specimen. The weighing machine used had following parameters:

Least Count – 0.001gm Max. Capacity – 5 Kg

## Diagram of Experimental Setup



*Fig-4.2 Experimental Setup –With Controller*



Fig-4.3 Experimental setup (Front View)



Fig-4.4 Experimental set up (Top View)



*Fig-4.6 Specimen*



## RESULTS AND DISCUSSION

Table 1: Variation in Two-body and Three Abrasive Wear of Aluminium alloy(6063) at 5N Load

<b>SET NO.1( 100rpm,5N )</b>					
TEST NO.	MASS BEFORE TEST (gm)	MASS AFTER TEST (TWO BODY test) (gm)	Wear by mass (Two BODY WEAR) gm	Mass after three body test (gm)	Wear by Mass(three Body wear
1.	54.921	54.458	0.463	54.344	0.114
2.	55.139	54.711	0.428	54.529	0.182
3.	53.673	53.215	0.458	53.020	0.195
4.	54.116	53.787	0.329	53.568	0.219
5.	53.545	53.326	0.219	53.200	0.126
MEAN			0.379		0.167

Table 2:

<b>SET NO.2( 150rpm,5N )</b>					
TEST NO.	MASS BEFORE TEST(gm)	MASS AFTER TEST (TWO BODY test) (gm)	Wear by mass gm	Mass after three body test	Wear by mass
1.	54.344	53.600	0.744	53.335	0.265
2.	54.529	53.955	0.574	53.670	0.285
3.	53.020	52.530	0.490	52.200	0.330



4.	53.568	52.830	0.738	52.535	0.295
5.	53.200	52.650	0.550	52.355	0.295
MEAN			0.619		0.294

Table 3:

**SET NO.3( 200RPM,5N )**

TEST NO.		MASS BEFORE TEST		mass after test		Wear by mass
WEAR BY MASS		mass after three				
		(gm)		(two body test)		
(gm)		body test(gm)				
1.	53.300	52.575	0.725	52.100	0.475	
2.	53.655	52.800	0.855	52.350	0.450	
3.	52.200	51.455	0.745	50.970	0.485	
4.	52.530	51.700	0.830	51.235	0.465	
5.	52.355	51.600	0.755	51.225	0.375	
mean			0.782		0.450	

**SET NO.4( 100RPM,10N )**

TEST NO.	MASS BEFORE TEST		mass after test		WEAR	Wear by mass
BY MASS	mass after					
	(gm)		(two body test)(gm)	(two body		
wear)gm	three body test					
1.	52.100	51.525	0.575	51.155	0.370	
2.	52.350	51.856	0.494	51.375	0.481	
3.	50.970	50.235	0.735	49.800	0.435	
4.	51.235	50.560	0.675	50.145	0.415	
5.	51.200	50.620	0.580	50.130	0.490	
mean			0.611		0.438	

**SET NO.5( 150RPM,10N )**

TEST NO.	MASS BEFORE TEST	mass after test	Wear by mass
WEAR BY MASS	mass after three (gm)	(two body test)(gm)	
(gm)	body test(gm)		
1.	51.155	50.550	0.605
2.	51.375	50.700	0.675
3.	49.800	49.224	0.576
4.	50.145	49.450	0.695
5.	50.130	49.635	0.495
mean		0.609	

**SET NO.6( 200RPM,10N )**

TEST NO.	MASS BEFORE TEST	mass after test	Wear by mas
WEAR BY MASs	mass after (gm)	(two body test)(gm)	
(gm)	three body(gm)		
1.	50.200	49.337	0.863
2.	50.350	49.755	0.595
3.	48.835	48.105	0.730
4.	49.100	48.322	0.778
5.	49.225	48.445	0.780
mean		0.749	

**SET NO.7( 100RPM,15N )**

TEST NO.		MASS BEFORE TEST		mass after	WEAR	Wear by mass
BY MASS		mass after				
		(gm)	(two body test)(gm)			
(gm)		three body test				
1.	48.900	48.230	0.670	47.620	0.610	
2.	49.225	48.665	0.560	48.223	0.442	
3.	47.554	46.882	0.672	46.260	0.622	
4.	47.883	47.120	0.763	46.570	0.550	
5.	47.900	47.220	0.680	46.600	0.620	
mean			0.669		0.568	

**SET NO.8( 150RPM,15N )**

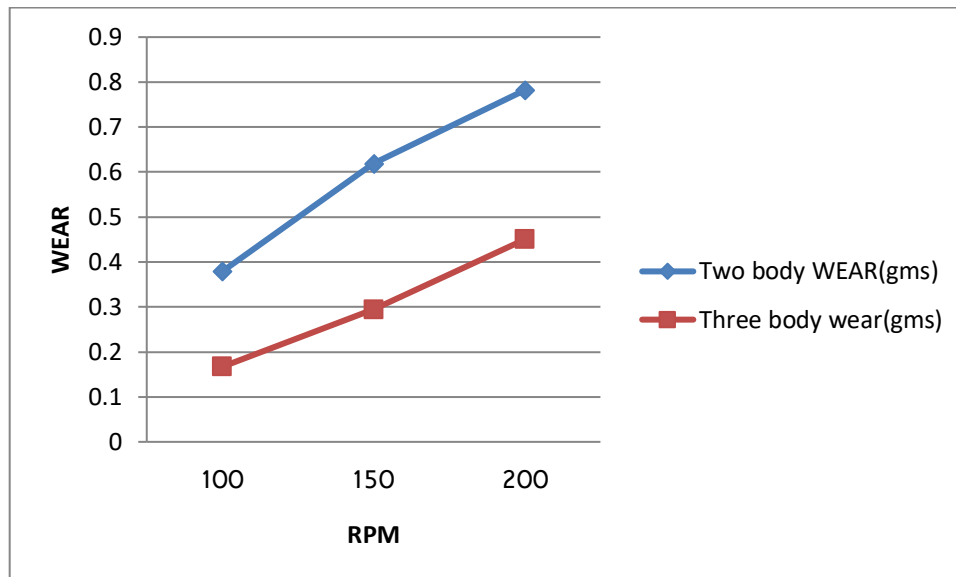
TEST NO.		MASS BEFORE TEST		mass after		Wear by mas
WEAR BY MASS		mass after				
		(gm)	(two body test)(gm)	(two		
body wear)gm		three body test				
1.	47.620	46.800	0.820	46.210	0.590	
2.	47.950	47.177	0.773	46.450	0.727	
3.	46.260	45.445	0.815	44.910	0.535	
4.	46.570	45.653	0.917	44.950	0.703	
5.	46.600	45.775	0.825	45.120	0.655	
mean			0.830		0.642	

**SET NO.9( 200RPM,15N )**

TEST NO.		MASS BEFORE TEST		mass after		WEAR		Wear by mass
BY MASS		mass after						
		(gm)		(two body wear)(gm)		(two body		
wear)gm	three body test							
1.	46.210	45.305	0.905	44.665	0.640			
2.	46.450	45.571	0.879	44.910	0.661			
3.	44.910	44.102	0.808	43.720	0.382			
4.	44.950	44.110	0.840	43.620	0.490			
5.	45.120	44.250	0.870	43.770	0.480			
mean			0.860		0.530			

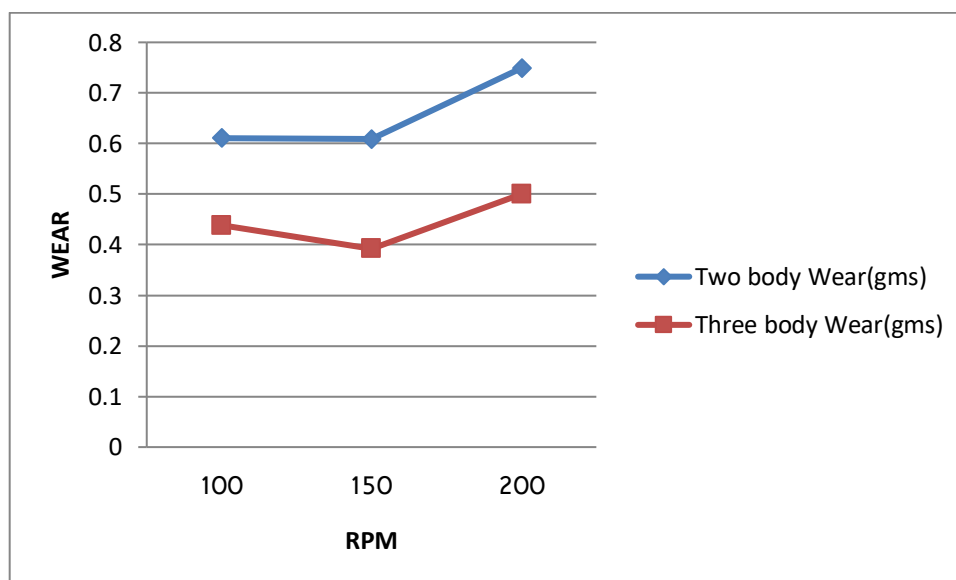
**At 5N- RPM vs. WEAR**

RPM	Two body WEAR(gms)	Three body wear(gms)
100	0.379	0.167
150	0.619	0.294
200	0.782	0.450



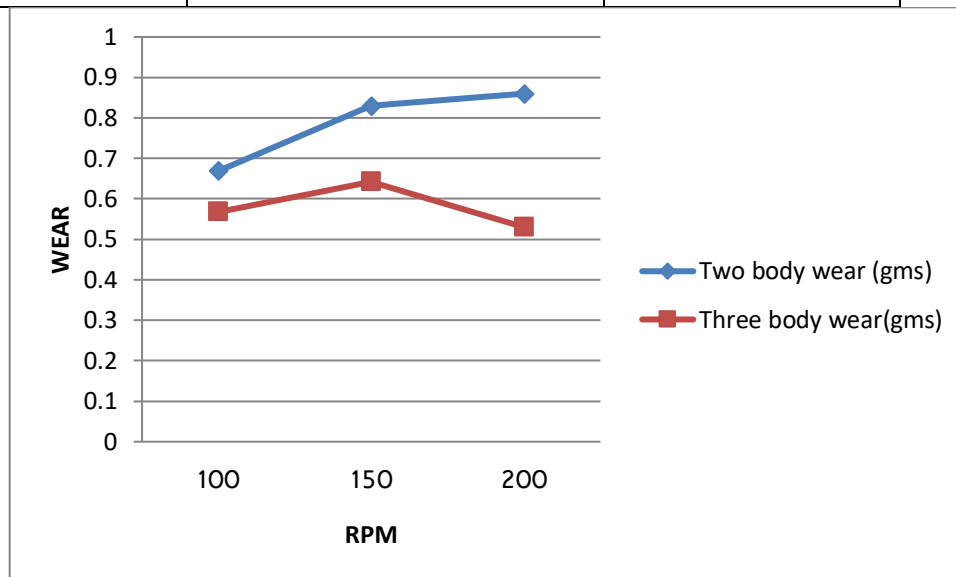
#### At 10 N- RPM vs. WEAR

rpm	Two body Wear(gms)	Three body Wear(gms)
100	0.611	0.438
150	0.609	0.392
200	0.749	0.500



### At 15 N- RPM vs. WEAR

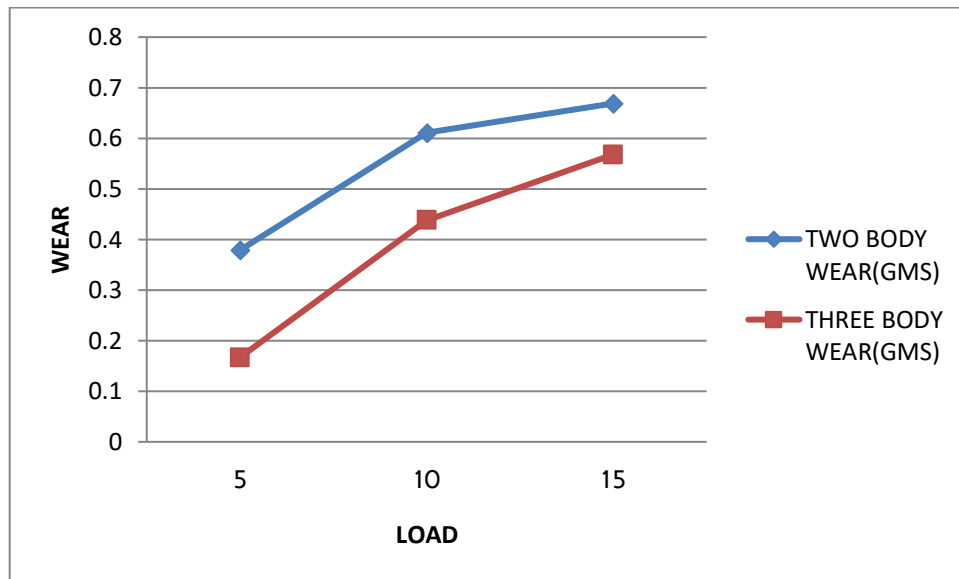
RPM	Two body wear (gms)	Three body wear(gms)
100	0.669	0.568
150	0.830	0.642
200	0.860	0.530



### Effect of Speed (rpm) On Abrasive Wear of ALUMINIUM AL6063 AT CONSTANT ANGULAR SPEED

#### LOAD VS WEAR (100RPM)

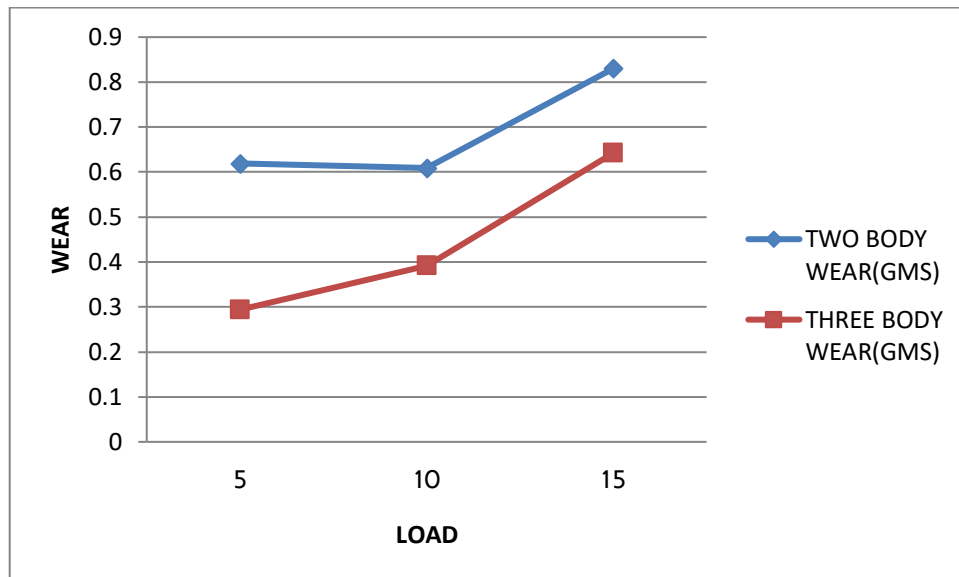
LOAD(N)	TWO BODY WEAR(GMS)	THREE BODY WEAR(GMS)
5	0.379	0.167
10	0.611	0.438
15	0.669	0.568



GRAPH

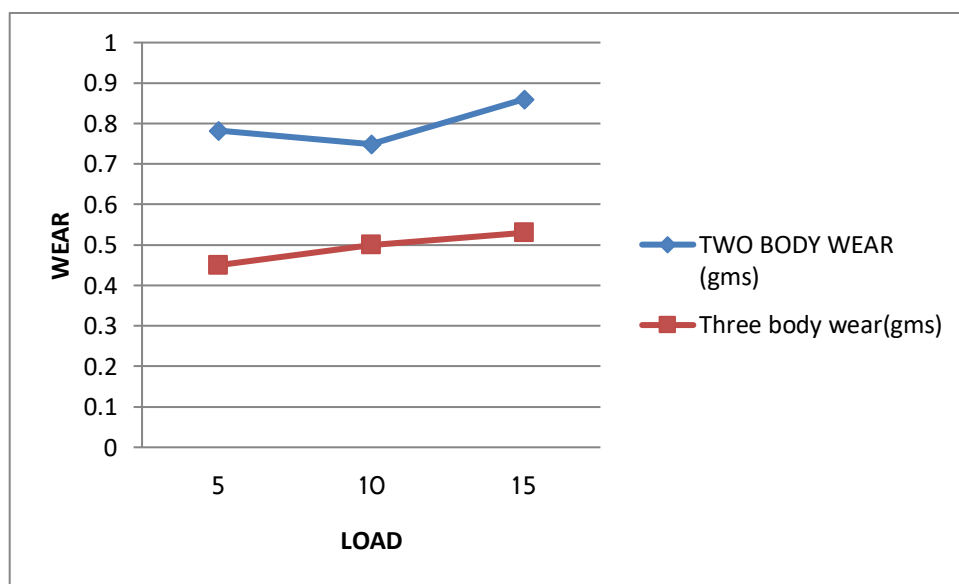
#### LOAD VS WEAR (150RPM)

LOAD(N)	TWO BODY WEAR(GMS)	THREE BODY WEAR(GMS)
5	0.619	0.294
10	0.609	0.392
15	0.830	0.642



**LOAD VS WEAR(200RPM)**

LOAD(N)	TWO BODY WEAR (gms)	Three body wear(gms)
5	0.782	0.450
10	0.749	0.500
15	0.860	0.530





## DISCUSSION

The main findings of the investigation have been listed out. The suggestion for the future work have also been indicated. The specimen do not get fresh abrasive surface, due to this wear resistance increases. Following results are discussed below:

### GRAPH-5.1

This graph shows RPM vs WEAR at 5 N. The wear loss increases while RPM increases. The graph is not linear in nature.

### GRAPH-5.2

This graph shows RPM vs WEAR at 10 N. The wear loss increases while RPM increases. The graph is not linear in nature.

### GRAPH-5.3

This graph shows RPM vs WEAR at 15 N. The wear loss increases while RPM increases. The graph is not linear in nature. In this graph wear loss is more as compare to graph 1 and graph 2.

### GRAPH-5.4

This graph shows LOAD vs WEAR at 5 N. The wear loss increases while load increases. The graph is linear in nature, this shows linear relationship between load and wear.

### GRAPH-5.5

This graph shows LOAD vs WEAR at 10 N. The wear loss increases while load increases. The graph is linear in nature , this shows linear relationship between load and wear.

#### GRAPH-5.6

This graph shows LOAD vs WEAR at 15 N. The wear loss increases while load increases. The graph is linear in nature , this shows linear relationship between load and wear. In this graph wear loss is more as compare to graph 1 and graph 2.

## CONCLUSION

It is concluded from the above discussion that wear is function of applied load. Initially, it was understood that wear depends upon applied load, surface parameters and mechanical properties such as hardness, toughness etc. Thus it can be concluded that:

- There is a linear relationship between wear and load
- The wear loss increases while load increases. Wear loss is more at 15 N load as compare to 5N and 10 N load.
- The wear loss increases while RPM increases.
- The wear loss in first minute is more as compare to last minute while increasing the RPM

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